



Chapter 25

Undamped coupled 2nd-order ODEs

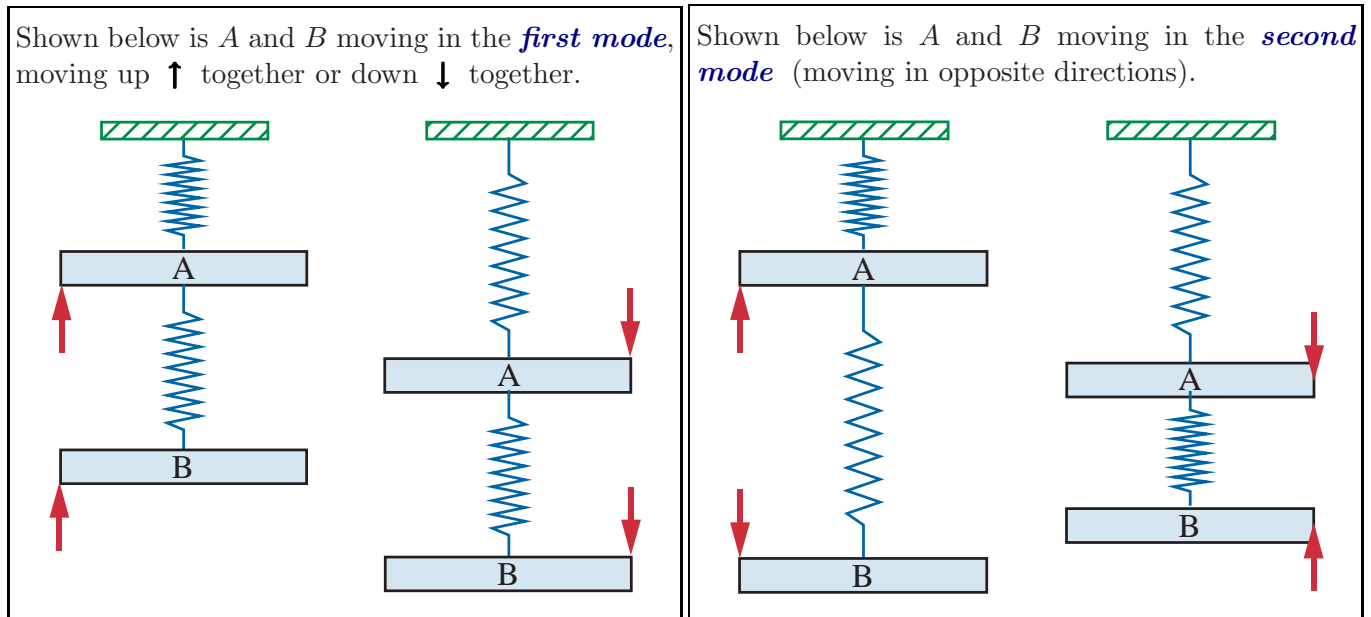
Summary (see examples in Hw 13)

Many physical phenomena are governed by a set of n **undamped**, coupled, linear, constant-coefficient, homogeneous ODEs. These include free vibrations of buildings, airplanes, automobiles, space structures, and molecules.

$$M \ddot{X} + K X = [0] \quad X = \begin{bmatrix} x_1(t) \\ x_2(t) \\ \dots \\ x_n(t) \end{bmatrix}$$

25.1 Physical insights into eigenvalues and eigenvectors of a slinky

Two long thin rulers (A and B) connected to a slinky (spring) are useful for demonstrating the physical significance of eigenvalues and eigenvectors.



Mode #	Eigenvalue (frequency)	Eigenvector (use + or 0 or -)	Potential Energy ($\frac{1}{2} k \delta^2$)	Kinetic Energy ($\frac{1}{2} m v^2$)	Total Energy Kinetic + Potential
1	small /large	$\begin{bmatrix} + \\ + \end{bmatrix}$ or $\begin{bmatrix} - \\ - \end{bmatrix}$	small /large one/two deformed spring min /max deformation	small /large	small /large
2	small/ large	$\begin{bmatrix} + \\ - \end{bmatrix}$ or $\begin{bmatrix} - \\ + \end{bmatrix}$	small/ large one/two deformed springs min / max deformation	small/ large	small/ large

Answers at www.MotionGenesis.com \Rightarrow [Textbooks](#) \Rightarrow [Resources](#).

25.2 Analytical solutions of *undamped, coupled*, ODEs

X	$n \times 1$ matrix of dependent variables
M, B, K	$n \times n$ matrices of constants
G	$n \times p$ matrix of constants
F	$p \times 1$ matrix of known functions of time $f_1(t), f_2(t), \dots, f_p(t)$.

An undamped set of n coupled, linear, constant-coefficient, **2nd-order ODEs** can be written in the matrix form

$$\begin{matrix} M & \ddot{X} & + & B & \dot{X} & + & K & X & = & G & F \\ (n \times n) & & & (n \times n) & & & (n \times n) & & & (n \times p) & (p \times 1) \end{matrix} \quad (1)$$

To solve equation (1), write $X(t) = X_h(t) + X_p(t)$ (the sum of a homogeneous and particular solution). The homogeneous solution $X_h(t)$ can be found by assuming a solution of the form shown in equation (2), where p is a yet-to-be-determined constant and U is a yet-to-be-determined **non-zero** $n \times 1$ matrix of constants.^a

$$X_h(t) = U e^{pt} \quad (2)$$

$$U \triangleq \begin{bmatrix} u_1 \\ u_2 \\ \vdots \\ u_n \end{bmatrix}$$

^aIt is reasonable to guess $X_h(t) = U e^{pt}$ because it worked for **uncoupled** linear ODEs. The matrix U is **non-zero** because $U = [0]$ produces the **trivial solution** $X_h(t) = [0]$, which is not a solution of interest (not what we are looking for).

Substituting/differentiating equation (2) into the homogeneous part of equation (1) ($F = [0]$), gives

Note: This proof mimics Section 23.1. Answers are at www.MotionGenesis.com \Rightarrow [Textbooks](#) \Rightarrow [Resources](#).

$$M \begin{bmatrix} p^2 U e^{pt} \end{bmatrix} + B \begin{bmatrix} p U e^{pt} \end{bmatrix} + K \begin{bmatrix} U e^{pt} \end{bmatrix} \stackrel{(1,2)}{=} [0] \quad \text{Factor on } U e^{pt}$$

$$\begin{bmatrix} p^2 M + p B + K \end{bmatrix} U e^{pt} = [0] \quad \text{Section 7.1 shows } e^{pt} \neq 0$$

$$\begin{bmatrix} p^2 M + p B + K \end{bmatrix} U = [0] \quad \text{Sometimes } B \approx [0] \text{ (see } \mathbf{undamped} \text{ below)}$$

$$\begin{bmatrix} p^2 M + K \end{bmatrix} U = [0] \quad \text{Define: } \lambda \triangleq -p^2 \quad p = \pm \sqrt{-\lambda}$$

$$\mathbf{Generalized eigen-equation} \quad \begin{bmatrix} -\lambda M + K \end{bmatrix} U = [0] \quad \Rightarrow \quad \det \begin{bmatrix} -\lambda M + K \end{bmatrix} = 0 \quad (3)$$

This eigen-equation is a **coupled** nonlinear algebraic equation. It has n equations and $n + 1$ unknowns in λ and U .

The related $\det[-\lambda M + K] = 0$ is **1 uncoupled** nonlinear algebraic equation with **1** unknown λ .

Equation (1) \Rightarrow (3) changes n coupled ODEs with n unknowns into a **eigen-equation**.

Undamped: It can be reasonable to set $B = [0]$ if:

- Damping is small.
- It is difficult to analytically or experimentally determine the elements of the damping matrix.
- In structural vibrations, damping is approximated with **modal damping**.

For eigen-equation (3) to produce a **non-zero** U (and **non-zero** X_h), the inverse of $[-\lambda M + K]$ must not exist. To see this, suppose $[-\lambda M + K]^{-1}$ does exist and multiply both sides of the eigen-equation by $[-\lambda M + K]^{-1}$, which produces $U = [0]$. For $U \neq [0]$, one must find **special** values of λ so $[-\lambda M + K]^{-1}$ does not exist. These **special** values $\lambda_1, \lambda_2, \dots, \lambda_n$ are called **eigenvalues**. For each λ_i , there is a corresponding **special** non-zero U_i called the **eigenvector** corresponding to λ_i . Note: The eigenvalue problem is a **special** nonlinear algebraic equation because the number of solutions is known apriori.

The following are equivalent statements about equation (4) and finding the **eigenvalues** λ :

- Find the values of λ which result in $U \neq [0]$.
- Find the values of λ so the matrix $[-\lambda M + K]$ is singular, i.e., $[-\lambda M + K]^{-1}$ does not exist.
- Find the values of λ so the determinant of $[-\lambda M + K]$ is zero, i.e., $\det[-\lambda M + K] = 0$

Computational notes: For small sets of coupled equations ($n \leq 3$), setting the determinate to zero is an effective way to determine λ - and this method works when M or K have symbolic (non-numeric) elements. When n is large, numerical eigenvalue algorithms are used to determine λ . Robust eigenvalue and eigenvector algorithms are relatively new (circa 1960) and are available in computer programs such as MATLAB[®] and MotionGenesis. If both M and K are symmetric and positive definite, the eigenvalues λ_i ($i=1, \dots, n$) are real positive numbers and eigenvectors U_i ($i=1, \dots, n$) are real (not complex), and there are special efficient algorithms for calculating λ_i and U_i (especially important when n is large).